

Answer Key

Keys and Splines

$$P = \frac{2\pi TN}{60} ; S_c = \frac{2T}{dbL} ; FS = \frac{0.6 S_y}{S_s}$$

Subst;

$$\frac{0.6 S_y}{FS} = \frac{2T}{dbL}$$

$$\frac{(0.6)(70000 \text{ KPa})}{4} = \frac{2T}{\left(3.14 \left(\frac{1 \text{ ft}}{12 \text{ in}}\right) \left(\frac{1 \text{ m}}{3.28 \text{ ft}}\right) (0.025 \text{ m})(0.05 \text{ m})\right)}$$

$$T = 0.500 \text{ kN-m}$$

Hence;

$$P = \frac{2\pi (0.500 \text{ kN-m})(300 \text{ rpm})}{60}$$

$$P = 19.9043 \text{ kW}$$

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For shear

$$S_s = \frac{2T}{dbL}$$

$$62000 \text{ kPa} = \frac{2(0.915 \text{ kN-m})}{(0.064 \text{ m})(0.012 \text{ m})L}$$

$$L = 0.03843 \text{ m}$$

$$L = 38.43 \text{ mm}$$

For compression/Tension:

$$S_c = \frac{4T}{dtL}$$

$$100000 \text{ kPa} = \frac{4(0.915 \text{ kN-m})}{(0.064 \text{ m})(0.016 \text{ m})L}$$

$$L = 0.0357 \text{ m}$$

$$L = 35.7 \text{ mm}$$

Hence;

choose : $L = 38.43 \text{ mm}$

Based on shearing in the key

$$\frac{0.6 S_y}{FS} = \frac{2T}{dbL}$$

Based on torsional stress:

$$\frac{0.6 S_y}{FS} = \frac{16T}{\pi d^3}$$

$$\frac{0.6 (60000 \text{ lb/in}^2)}{4.0} = \frac{16T}{\pi \left(\frac{5.2}{25.4} \text{ in}\right)^3}$$

$$T = 15162.8654 \text{ lb-in}$$

Hence;

$$\frac{0.6 S_y}{4.0} = \frac{2 (15162.8654 \text{ lb-in})}{\left(\frac{5.2}{25.4} \text{ in}\right) \left(\frac{3}{4} \text{ in}\right) (4.5 \text{ in})}$$

$$S_y = 29.2602 \text{ KSI}$$

for the shearing stress at key

$$S_s = \frac{2T}{dbL}$$

$$220000 \text{ KPa} = \frac{2T}{(0.10\text{m})(0.03\text{m})(0.100\text{m})}$$

$$T = 59.4 \text{ kN}\cdot\text{m}$$

from;

$$T_{\text{pulley}} = T_{\text{key}}$$

$$F_{\text{pulley}} \left(\frac{D_{\text{pulley}}}{2} \right) = T_{\text{key}}$$

$$F_{\text{pulley}} \left(\frac{1\text{m}}{2} \right) = 59.4 \text{ kN}\cdot\text{m}$$

$$F_{\text{pulley}} = 118.8 \text{ kN}$$

$$S_c = \frac{F_c}{A_c} = \frac{F_a}{\left(\frac{t}{2}\right)L} ; L = 5t$$

$$420000 \text{ kPa} = \frac{150 \text{ kN}}{\left(\frac{t}{2}\right)(5t)}$$

$$t = 0.01195 \text{ m}$$

$$t = 11.95 \text{ mm}$$

$$S_c = \frac{4T}{d t L}$$

$$450\,000 \text{ kPa} = \frac{4(2.5 \text{ kN-m})}{(0.05 \text{ m}) t (0.022 \text{ m})}$$

$$t = 0.02020 \text{ m}$$

$$t = 20.20 \text{ mm}$$

For Line Shafts

$$P = \frac{D^3 N}{80}$$

$$(18 \text{ kw}) \left(\frac{1 \text{ hp}}{0.746 \text{ kw}} \right) = \frac{D^3 (360 \text{ rpm})}{80}$$

$$D = 1.4930 \text{ in (shaft diameter)}$$

for the key

$$S_s = \frac{F_s}{A_s} = \frac{F_s}{bL} ; b = \frac{1}{4} d$$

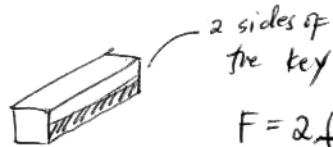
$$250\,000 \text{ kPa} = \frac{F_s}{\frac{1}{4} (1.4930 \text{ in}) (1 \text{ in}) \left(\frac{1 \text{ ft}^2}{144 \text{ in}^2} \right) \left(\frac{1 \text{ m}}{3.28 \text{ ft}} \right)^2}$$

$$F_s = 60.2331 \text{ kN}$$

$$S_c = \frac{4T}{d t L}$$

$$28000 \frac{\text{lb}}{\text{in}^2} = \frac{4 T}{(2.5 \text{ in}) \left(\frac{7}{16} \text{ in} \right) (3.5 \text{ in})}$$

$$T = 26796.875 \text{ in-lb}$$



$$F = 2fN ; P = \frac{2\pi TN}{60}$$

$$120 \text{ kW} = \frac{2\pi T (1300 \text{ rpm})}{60}$$

$$T = 0.8815 \text{ kN-m}$$

Then;

$$T = F \left(\frac{d}{2} \right)$$

$$0.8815 \text{ kN-m} = F \left(\frac{0.055 \text{ m}}{2} \right)$$

$$F = 32.0536 \text{ kN}$$

Hence; the force required to remove shaft from hub

$$F = 2(0.50)(32.0536 \text{ kN})$$

$$F = 32.0536 \text{ kN}$$

$$P = \frac{2\pi TN}{60}$$

$$200 \text{ kW} = \frac{2\pi T(420 \text{ rpm})}{60}$$

$$T = 4.5473 \text{ kN-m}$$

and from,

$$S_s = \frac{2T}{dbL}$$

$$220000 \text{ kPa} = \frac{2(4.5473 \text{ kN-m})}{(0.075 \text{ m})(0.018 \text{ m})L}$$

$$L = 0.0306 \text{ m}$$

$$L = 30.6 \text{ mm}$$

$$A_c = 2.5 \text{ in}^2$$

$$A_c = t L N_s$$

$$(2.5 \text{ in}^2) = (0.100)(2.25 \text{ in})(6)$$

$$D = 1.8519 \text{ in}$$

$$A_s = 3.5 \text{ in}^2$$

$$A_s = b L N_s$$

$$(3.5 \text{ in}^2) = (0.250)(2.25 \text{ in})(6)$$

$$D = 1.0370 \text{ in}$$

choose: $D = 1.0370 \text{ in}$

4 spline shaft

$$d = 0.85D = 0.85(1.5 \text{ in}) = 1.275 \text{ in}$$

$$b = 0.241D = 0.241(1.5 \text{ in}) = 0.3615 \text{ in}$$

$$t = 0.075D = 0.075(1.5 \text{ in}) = 0.1125 \text{ in}$$

$$S_s = \frac{F}{A_s} = \frac{F}{bLN_s}$$

$$900 \frac{\text{lb}}{\text{in}^2} = \frac{F}{(0.3615 \text{ in})(2 \text{ in})(4)}$$

$$F = 2602.8 \text{ lbs}$$

$$S_c = \frac{4T(1.1)}{(D+d)(t)(L)N_c} ; HP = \frac{2\pi TN}{33000}$$

$$120HP = \frac{2\pi T(1850rpm)}{33000}$$

$$T = 340.6776 \text{ lb-ft}$$

Subst:

$$S_c = \frac{4(340.6776 \text{ lb-ft})\left(\frac{12 \text{ in}}{1 \text{ ft}}\right)(1.1)}{(2.5 \text{ in} + 2 \text{ in})(0.1875 \text{ in})(0.18)(2.5 \text{ in})(6)}$$

$$S_c = 7895.8693 \text{ psi}$$

$$HP = \frac{2\pi TN}{33000}$$

$$(100 \text{ kW}) \left(\frac{1 \text{ hp}}{0.746 \text{ kW}} \right) = \frac{2\pi T (2000 \text{ rpm})}{33000}$$

$$T = 352.0183 \text{ lb-ft}$$

$$\text{from: } T_s = \frac{T}{N_s} \quad ; \quad T_s = F \left(\frac{D+d}{4} \right)$$

$$F \left(\frac{D+d}{4} \right) = \frac{T}{N_s}$$

$$F = \frac{4T}{(D+d)N_s} = \frac{4(352.0183 \text{ lb-ft}) \left(\frac{12 \text{ in}}{1 \text{ ft}} \right)}{(1.8 + 1.62) \text{ in} (8)}$$

$$F = 617.5759 \text{ lbs (Force per spline)}$$

$$HP = \frac{2\pi TN}{33000} \quad ; \quad S_c = \frac{4T}{(D+d)tLN_c}$$

$$820 \frac{\text{lb}}{\text{in}^2} = \frac{4T}{(2.5 + 2)\text{in}(0.25\text{in})(1.625\text{in})(6)}$$

$$T = 2248.5938 \text{ in-lb}$$

Subst:

$$HP = \frac{2\pi (2248.5938 \text{ in-lb}) \left(\frac{1 \text{ ft}}{12 \text{ in}}\right) (3000 \text{ rpm})}{33000}$$

$$HP = 107.0328 \text{ hp}$$

$$S_c = \frac{F}{A_c} = \frac{F}{t L N_s}$$

$$\frac{800 \text{ lb}}{\text{in}^2} = \frac{F}{(0.1750 \text{ in})(2.5 \text{ in})(4)}$$

$$F = 1400 \text{ lbs}$$

Based on Shearing of Spline

$$T_s = \frac{T}{N_s} ; F = \frac{2T_s}{d}$$

$$T = \frac{F d N_s}{2} = \frac{(2300 \text{ lbs})(2.38 \text{ in})(6)}{2} = 16422 \text{ in-lbs}$$

Substi

$$HP = \frac{2\pi TN}{33000} = \frac{2\pi (16422 \text{ in-lb}) \left(\frac{1 \text{ ft}}{12 \text{ in}} \right) (2600 \text{ rpm})}{33000}$$

$$HP = 677.4607 \text{ hp}$$

Based on Compression of Spline

$$T_s = \frac{T}{N_s} ; F = \frac{T_s}{\left(\frac{D+d}{4}\right)}$$

$$T = F \left(\frac{D+d}{4}\right) N_s = 2000 \text{ lbs} \left[\frac{(1.5 + 1.365) \text{ in}}{4} \right] (10)$$

$$T = 14325 \text{ in-lb}$$

Based on Shearing of Spline

$$S_s = \frac{F}{A_s}$$

$$S_s = \frac{F}{b L N_s} = \frac{2400 \text{ lbs}}{(0.625 \text{ in})(2 \text{ in})(4)}$$

$$S_s = 400 \text{ psi}$$

Based on Compression of Spline

$$S_c = \frac{F}{A_c} = \frac{F}{t L N_s} = \frac{2200 \text{ lbs}}{(0.1 \text{ in})(1.5 \text{ in})(6)}$$

$$S_c = 2444.4444 \text{ psi}$$

24 to 27

$$d = 0.8/D = 0.8/(2.25 \text{ in}) = 1.8225 \text{ in}$$

$$b = 0.156 D = 0.156(2.25 \text{ in}) = 0.351 \text{ in}$$

$$t = 0.095 D = 0.095(2.25 \text{ in}) = 0.2138 \text{ in}$$

$$L = 1.625 \text{ in}$$

$$N = 2800 \text{ rpm}$$

$$S_c = 850 \text{ psi}$$

$$a) A_c = tL = (0.2138 \text{ in})(1.625 \text{ in})$$

$$A_c = 0.3474 \text{ in}^2$$

$$b) S_c = \frac{F}{A_c} = \frac{F}{A_c N_s}$$

$$850 \frac{\text{lb}}{\text{in}^2} = \frac{F}{0.3474 \text{ in}^2}$$

$$F = 295.29 \text{ lbs}$$

$$c) T = T_s N_s = \frac{F_c d}{2} N_s$$

$$= (295.29 \text{ lbs}) \left(\frac{1.8225 \text{ in}}{2} \right) (10)$$

$$T = 2690.8301 \text{ in-lbs}$$

$$d) \text{HP} = \frac{2\pi T N}{33000} = \frac{2\pi (2690.8301 \text{ in-lbs}) \left(\frac{1 \text{ ft}}{12 \text{ in}} \right) (2800 \text{ rpm})}{33000}$$

$$\text{HP} = 119.5443 \text{ hp}$$